

IN THE UNITED STATES PATENT AND TRADEMARK OFFICE

PATENT APPLICATION

For

REDUCED WEIGHT GUIDE LINK

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Reduced Weight Guide link

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Cross Reference to Related Applications

The present application is a continuation-in-part of U.S. Patent Application ser. no. 09/883,080, filed June 15, 2001, attorney's docket 2229/128, which is a continuation-in-part of U.S. Patent Application ser. no. 09/335,392, attorney's docket 2229/104, filed June 17, 1999 and now issued as U.S. Patent ser. no. 6,253,550, which are all herein 10 incorporated by reference.

Technical Field

The present invention pertains to improvements to an engine and more particularly to improvements relating to mechanical components of a Stirling cycle heat engine or refrigerator which contribute to increased engine operating efficiency and 15 lifetime.

Background of the Invention

A major problem encountered in the design of certain engines, including the compact Stirling engine, is that of the friction generated by a sliding piston resulting from misalignment of the piston in the cylinder and lateral forces exerted on the piston 20 by the linkage of the piston to a rotating crankshaft. Referring now to the engine portion 100 shown in Fig. 1, as described in U.S. Patent ser. no. 6,253,550, it is known that in order to reduce the lateral forces on the piston, a guide link 103 may be used as a guidance system to take up lateral forces while keeping the motion of piston 14 constrained to linear motion. In a guide link design, a connecting rod is replaced by the 25 combination of guide link 103 and a connecting rod 105. Guide link 103 is rotatably connected to the connecting rod 105 at a crank-pin 108. Guide link 103 is aligned with the wall 101 of piston cylinder 14 and is constrained to follow linear motion by a set of guide wheels 109, 111. A guide link will generally increase the size of the crankcase because the guide link must be of sufficient length that when the piston is at its 30 maximum extension into the piston cylinder, the guide link extends beyond the piston cylinder so that the guide wheels maintain contact and alignment with the guide link.

However, the folded guide link, as disclosed in U.S. Patent ser. no. 6,253,550, can reduce the size of the crankcase.

Guide link **103** is subject to wear at those points where rollers **109** and **111** contact the guide link. Constructing the guide link of a durable material, for example, a **5** hardened metal, may significantly increase the weight of the engine. For certain applications, such as transportation, lighter weight engines are preferable.

Summary of the Invention

In accordance with an aspect of the invention, a method of manufacturing a reduced-weight guide link for a thermal cycle engine is provided. The method includes **10** fabricating a guide link from a lightweight material; providing inserts of a material that is more wear-resistant than the guide link, and securing the inserts to the guide-link as guideways for the guide wheels. In this way the guide link may better withstand the constant wear from the guide wheel as the wheel traverses the guide-way, while the engine reciprocates. The balance of the guide-link may be fabricated from lighter weight **15** materials that need not withstand the constant wear of the guide wheels. In this way, a lighter guide-link may be fabricated and other components within the engine may be correspondingly reduced in weight, achieving a substantial weight reduction for the engine.

The guideway inserts may be fabricated from a hardened metal or a ceramic **20** material. The insert may be coated to further increase wear resistance. The shape of the insert may be any shape that mates with a guide wheel surface shape. In specific embodiments of the invention, the insert may be shaped as a cylinder, may have a square cross-section or, may be V-shaped in the faces that the insert presents to the guide wheel.

Brief Description of the Drawings

25 The invention will be more readily understood by reference to the following description, taken with the accompanying drawings, in which:

Fig. **1** is a cross-sectional view of a folded guide link linkage for an engine;

Fig. **2** is a perspective view of an embodiment of a reduced weight guide link;

Fig. **3** is a perspective view of the guide link of Fig. **2** shown without guide

30 wheels; and

Fig. 4 is a cross sectional view of alternative insert and guideway shapes.

Detailed Description of Preferred Embodiments

An improvement to a guide link for a reciprocating engine, according to an embodiment of the invention, is shown in Fig. 2. This guide link may be used, for 5 example, in the embodiment of Fig. 1 in place of the guide link 103. The guide link 212 transfers axial loads from the crank-pin 208 to the piston connected at the top of the guide link 212. The guide link also transfers the transverse loads from the crank-pin 208 to the guide wheels 210. In addition to carrying the loads, the guide link must provide a hard and wear-resistant surface to contact the guide wheels. Although lightweight alloys 10 such as aluminum, magnesium or titanium can carry the loads, these materials would quickly wear away under the guide wheels. In this embodiment, two guideway inserts 214 are installed in a guide link 212 that may be fabricated from material different from that used in the inserts. The guideway inserts 214 resist wear caused by the guide wheels 220, while the guide link carries the load between the crank pin 208 and the piston. (One 15 of the insert is not shown in Fig. 2.) This design separates the load carrying function from the wear function, allowing each to be optimized. Thus, a lighter weight material may be used for the guide link 212, than would be possible if the guide link were of unitary design.

A number of alternative cross-sections may be used for the guide wheels 210 and 20 guideway inserts 214. For example, the cross-sectional alternatives shown in Fig. 4 may be used. In a specific embodiment of the invention, the guide wheels 210 are commercially available bearings with circular grooves 220 on the outside diameter. Cylindrical guide-way inserts 214 are secured to the guide link 212 by screws 216 and an end plate 218. These inserts are shown in cross-section in Fig. 4. In an alternative 25 embodiment, commercially available bar-stock is used for the guideway inserts 314 and is mated with a "V"-wheel for the guide wheels 310, as shown in Fig. 4. This insert has a square cross-section. In a further alternative embodiment, custom guideway inserts 414 are paired with "V" wheels for the guide wheels 410. These guideway inserts 414 present a V-shaped face to the guide wheels. The insert face(s) that contact the guide link may be a partial cylinder in cross-section or another shape. The term "V-shaped 30 insert" as used in this specification and in any appended claims will mean an insert which presents a V-shaped face to a guide wheel and presents an arbitrarily shaped face

to a guide link. In a further alternative embodiment, "V"-shaped guideway inserts **514** are also paired with "V" wheels for the guide wheels **510**. The inserts **514** slip over tabs **516** extending from the guide link **212**. These inserts **514** may be secured to the guide link by an endplate **218** that may be secured to the guide link by screws **216**.

5 In a preferred embodiment, chrome steel hardened to 59 to 65 Rockwell Hardness C is used for the guideway inserts. In this specification and in any appended claims, unless context requires otherwise, the term "hardened" as applied to a material shall indicate material hardened to 55C or greater on the Rockwell Hardness scale. In other embodiments, other materials including tool steels and other wear-resistant materials
10 may be used for the insert. In order to further increase wear resistance, the guideway inserts **214** may be coated with extremely hard materials. In preferred embodiments, coatings such as TiN (titanium nitride) and DLC (diamond like carbon) may be used.

15 Reducing the weight of any one reciprocating part of a guide link drive can reduce the total drive weight by a factor of four. This four-fold reduction comes from the balancing requirements. The rotating counter weights may be lightened to match the lightened guide link and the perpendicular piston and its counterweight may be equally reduced to provide a balanced drive. Embodiments of the invention may be used advantageously to reduce the weight of the engine for applications where engine weight is important.

20 The devices and methods described herein may be applied in other applications besides the Stirling engine in terms of which the invention has been described. The described embodiments of the invention are intended to be merely exemplary and numerous variations and modifications will be apparent to those skilled in the art. All such variations and modifications are intended to be within the scope of the present
25 invention as defined in the appended claims.